Study on ADRC Parameter Optimization Using CPSO for Clamping Force Control System

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Clamping force control system is essential for clamping tasks that require high precision. In this paper, Active Disturbance Rejection Controller (ADRC) is applied for clamping force control system, aiming to achieve higher control precision. Furthermore, the CPSO-ADRC system is proposed and implemented by optimizing the critical parameters of ordinary ADRC using chaos particle swarm optimization (CPSO) algorithm. To verify the effectiveness of CPSO-ADRC, Particle Swarm Optimization - ADRC (PSO-ADRC) is introduced as a comparison. The simulation results show that the CPSO-ADRC can effectively improve the control quality with faster dynamic response and better command tracking performance compared to ordinary ADRC and PSO-ADRC.

1. Introduction

Clamping mechanism has been widely used for workpiece processing in the industrial production [1]. Therefore, clamping force should be accurately regulated to prevent damage to the workpiece. The regulation of clamping force highly relies on the quality of the clamping force controller.

Active disturbance rejection controllers (ADRCs) not only inherit the advantages of conventional PID controller, but also make up for the shortcomings, which attracts special attention. Numerous studies have been conducted on the application of ADRC. For example, Tao et al. utilized ADRC to achieve safe and accurate homing of powered parafoils [2]. Zhang and Chen applied ADRC to achieve high precision tracking control of CNC machine tool feed drives [3]. Liu et al. investigated the pitch axis control of satellite camera based on a novel active disturbance rejection controller [4]. Shen et al. tried to solve the control problem of Autonomous Underwater Vehicle (AUV) by designing an active disturbance rejection controller for diving [5].

Though ADRC has many merits compare to conventional PID controller, very limited existence of its application can be found due to its complicated parameter adjustment. To overcome this problem, key parameters of ADRC are usually adjusted in two stages. In the first stage, parameters are roughly adjusted to estimate the ballpark ranges. In the second stage, parameters are roughly adjusted to estimate the ballpark ranges. In the second stage, further optimization to the parameters with specific algorithm is done to determine the precise adjustment. Abundant studies have been conducted for the optimization of ADRC parameters. Observingly, among the optimization algorithms, Genetic Algorithm (GA) [6, 7] and Particle Swarm Optimization (PSO) [8, 9] have been widely used in the optimization process to determine optimal ADRC parameters. Particularly, there are improved variants of GA and PSO algorithms that have even better optimization performance. For instance, the improved GA based on time scale model identification was presented to enable ADRC parameters optimization in a large scope [10]. The adaptive genetic algorithm (AGA) was proposed to solved the adjustment problem towards ADRC systems with overabundant number of parameters [11].
Improved PSO algorithms also showed superior performances on ADRC parameter optimization. An immune binary-state particle swarm optimization algorithm (IBPSO) was proposed and successfully applied to optimize the parameters of ADRC for chaotic system [12]. A chaos particle swarm optimization (CPSO) algorithm was introduced to conduct optimization and getting optimal parameters for ADRC of an antisynchronizing different chaotic systems [13].

In this paper, ADRC is implemented to clamping force control system with consideration of its superior advantage on robustness. In order to obtain better performance for clamping force control, further parameter optimization of ADRC has been conducted. Instead of ordinary PSO algorithm, CPSO algorithm is applied to this task for its better performance that can converge easier and avoid falling into local optimum.

This paper is organized as follows: Section 2 presents the work of establishing mathematical model of clamping force control; Sections 3 and 4 describe the ADRC and CPSO-ADRC design for clamping force control, respectively; Section 5 shows simulation experiments to verify the effectiveness of CPSO-ADRC with PSO-ADRC as comparison and Section 6 proposes some concluding remarks and prospects.

2. System Control Model

As shown in Figure 1, the diagram of clamping mechanism is drawn. When the middle rod moves to the left, the two clamping blocks on both sides will move close to each other and thus clamp the workpiece. As shown in Figure 2, $\theta_i$ is the servo motor rotation angle and $F$ is the clamping force acting on the clamped workpiece. To achieve the clamping force control, the mathematical model of clamping force control system is going to be built in the first step.

The equivalent torque balance equation for motor axis (see Figure 1) can be described as follows:

$$J \frac{d^2 \theta_m(t)}{dt^2} + B \frac{d \theta_m(t)}{dt} + i \cdot T_L(t) + K \left[ \theta_m(t) - \theta_i(t) \right]$$

$$+ T_D(t) = 0,$$

where $J$, $B$, $K$ are the equivalent total moment of inertia, equivalent total viscous damping coefficient, equivalent total stiffness coefficient, respectively, $i$ is the transmission ratio of gear reducer, $T_i(t)$ is the equivalent load torque, $\theta_i(t)$ is the input angle of servo motor, $\theta_m(t)$ is the motor shaft equivalent angle, $\theta_m(t) - \theta_i(t)$ is the shaft relative rotation angle, $T_D(t)$ is the sum of the uncertainty and disturbance torque, and $\theta_m$ is related to the middle rod displacement $x_a(t)$ as

$$\theta_m(t) = \frac{1}{2\pi} i \frac{2\pi}{L} x_a(t).$$

Equivalent load torque can be expressed as

$$T_L(t) = \frac{L}{2\pi \mu} + \Delta k F(t),$$

where $F(t)$ is the clamping force, $L$ is the lead-screw pitch, $\mu$ is the reverse stroke efficiency of ball screw, $\Delta k F$ is the uncertain part of equivalent load torque, and $\Delta k < L/4\pi\mu$.

According to the geometric relationship shown in Figure 1, then (4) can be obtained as

$$\tan \gamma = \frac{x_a(t)}{2x_a(t)},$$

where $x_a$ is the total stroke length of two clamp blocks. Then clamping force is determined as

$$F(t) = \frac{k x_a(t)}{2},$$

where $k$ is the stiffness of elastic material.

Substituting (2), (3), (4), and (5) in (1) and developing the mathematical expression, the model is now

$$J \frac{d^2 F(t)}{dt^2} + B \frac{d F(t)}{dt} + \frac{i^2 Lk \tan \gamma}{2\pi} \cdot T_L(t) + KF(t)$$

$$+ \frac{iLk \tan \gamma}{2\pi} T_D(t) = \frac{iLk \tan \gamma}{2\pi} K\theta_i(t).$$
Table 1: Clamping force control system’s parameters.

<table>
<thead>
<tr>
<th>Parameters</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>( J )</td>
<td>kg-mm(^2)</td>
<td>( 1.0 \times 10^5 )</td>
</tr>
<tr>
<td>( B )</td>
<td>N-mm-s/rad</td>
<td>10</td>
</tr>
<tr>
<td>( K )</td>
<td>N-mm/rad</td>
<td>( 2.0 \times 10^4 )</td>
</tr>
<tr>
<td>( L )</td>
<td>mm</td>
<td>5</td>
</tr>
<tr>
<td>( i )</td>
<td>/</td>
<td>0.32</td>
</tr>
<tr>
<td>( \gamma )</td>
<td>rad</td>
<td>( \pi/9 )</td>
</tr>
<tr>
<td>( k )</td>
<td>N/mm</td>
<td>10</td>
</tr>
<tr>
<td>( \mu )</td>
<td>/</td>
<td>0.85</td>
</tr>
</tbody>
</table>

This model can be expressed as a second-order system, where the state variable \( x_1 = F \), and

\[
\begin{align*}
\dot{x}_1 &= x_2, \\
\dot{x}_2 &= -a_1 x_1 - a_2 x_2 + gu(t) - d(t),
\end{align*}
\tag{7}
\]

where

\[
\begin{align*}
a_1 &= \frac{K}{J} + \frac{i^2 L k \tan \gamma}{2\pi J} \left( \frac{L}{2\pi \mu} + \Delta k \right) \\
a_2 &= \frac{B}{J} \\
g &= \frac{i L k \tan \alpha}{\pi J} K \\
d(t) &= \frac{i L k \tan \alpha}{\pi J} T_D(t).
\end{align*}
\tag{8}
\]

As \( K/J \gg (i^2 L k \tan \gamma/2\pi J)(L/2\pi \mu + \Delta k) \), therefore \( a_1 = K/J \). The parameters of clamping force control system are displayed in Table 1; then after calculation, we can obtain that \( a_1 = 200, a_1 = 0.1, g = 185.37 \). And to test the robustness of this system, \( d(t) \) is designed with a Gaussian function (Figure 3):

\[
d(t) = 100 \exp \left( -\frac{(t-a_1)^2}{2b_1^2} \right),
\tag{9}
\]

where \( c_1 = 5, b_1 = 0.5 \).

3. ADRC Design

ADRC can actively detect disturbance signal from the input and output signals of controlled object and then eliminate the disturbance before it impacts the system by adjusting the control signal. ADRC has three main components \[14\], tracking differential (TD), extended state observer (ESO), and nonlinear state error feedback (NLSEF). As shown in Figure 4, the ADRC structure for clamping force control system is designed.

Generally, the typical second-order ADRC can be defined as follows. TD is used for arranging the transient process, which can be expressed as

\[
\begin{align*}
\dot{v}_1 &= v_2, \\
\dot{v}_2 &= f(s,t)(v_1 - v_0, v_2, r, h),
\end{align*}
\tag{10}
\]

where \( v_0 \) is the input signal, \( v_1 \) is the tracking signal of \( v_0 \), \( v_2 \) is the differential signal of \( v_1 \), \( r \) is the parameter which determines the tracking speed, and \( h \) is the sampling step length.

ESO is the key component of ADRC, which is employed to monitor the output and predict the real-time state of clamping force control system to achieve disturbance compensation; it can be expressed as

\[
\begin{align*}
e &= z_1 - y \\
\dot{z}_1 &= z_2 - \beta_{01} e \\
\dot{z}_2 &= -\beta_{02} f_{al}(e, \alpha_1, \eta) + bu \\
\dot{z}_3 &= -h \times \beta_{03} f_{al}(e, \alpha_2, \eta),
\end{align*}
\tag{11}
\]

where \( y \) is the system output, \( z_1 \) is the tracking signal of \( y \), \( e \) is the deviation signal, \( z_2 \) is the differential signal of \( z_1 \), \( z_3 \) is the tracking signal of system disturbance, \( \beta_{01}, \beta_{02}, \) and \( \beta_{03} \) are gain coefficients relating to \( e, u \) is the control input, and \( \eta \) is the filtering factor.

Unlike traditional PID that uses linear strategy, NLSEF is a nonlinear control strategy which can enhance the precision for clamping force control system. It can be expressed as

\[
\begin{align*}
e_1 &= v_1 - z_1 \\
e_2 &= v_2 - z_2 \\
u_0 &= \beta_1 f_{al}(e_1, \alpha_{01}, \eta) + \beta_2 f_{al}(e_2, \alpha_{02}, \eta) \\
u &= u_0 - \frac{z_3}{b},
\end{align*}
\tag{12}
\]

Figure 3: \( d(t) \) varying with time.
where $e_1$ is the system error, $e_2$ is the differential of system error, $\beta_1$ and $\beta_2$ are gain coefficients, and $z_3/b$ is the total disturbance compensation of system.

The nonlinear function $f_{st}(x_1, x_2, r, h)$ can be expressed as

$$
u = f_{st}(x_1, x_2, r, h)$$
$$d = rh,$$
$$d_0 = hd,$$
$$y = x_1 + hx_2,$$
$$a_0 = \sqrt{d^2 + 8r|y|}$$

$$(13)$$

$$a = \begin{cases} 
\frac{x_2 + \left(\frac{a_0 - d}{2}\right) \times \text{sgn} \left( y \right)}{y} & |y| > d_0 \\
\frac{x_2 + \left(\frac{y}{h}\right)}{y} & |y| \leq d_0
\end{cases}$$

$$f_{st} = \begin{cases} 
\frac{-ra}{d} & |a| \leq d \\
-r\text{sgn} \left( a \right) & |a| > d
\end{cases}$$

The nonlinear function $f_{al}(e, \alpha, \eta)$ can be defined as

$$f_{al}(e, \alpha, \eta) = \begin{cases} 
|e|^{\alpha} \text{sgn} \left( e \right) & |e| > \eta \\
\frac{e}{\eta^{1-\alpha}} & |e| \leq \eta \quad \eta > 0
\end{cases}$$

$$(14)$$

where $\alpha$ is the nonlinear factor. To achieve higher control precision, it is necessary to further optimize the parameters of TD, ESO, and NLSEF. There are two parameters in TD, $[h, r]$, six in ESO, $[\alpha_{01}, \alpha_{02}, \beta_{01}, \beta_{02}, \beta_{03}, b]$, and five in NLSEF, $[\alpha_1, \alpha_2, \beta_1, \beta_2, \eta]$. In this paper, these parameters are divided into two groups, which are common parameters and key parameters. The common parameters and initial key parameters are determined based on empirical experience, which can be seen in Tables 2 and 3, respectively.

4. CPSO-ADRC Design

CPSO algorithm inherited the advantage of fast convergence from PSO algorithm, and to further avoid falling into local optimum situation, it also adopted CO (Chaos Optimization) algorithm for random traversal. Combining the two gives CPSO great advantage in practice.

As shown in Figure 5, flow chart of CPSO algorithm is drawn, where $N$ is the number of iterations, pbest is the current optimal solution, and gbest is the global optimal solution. After current optimal solution is obtained by PSO algorithm, further search in the nearby region of current optimal solution by CO algorithm will be continually conducted.

The fitness function $J$ is designed as follows:

$$J = \int_{0}^{\infty} \left( \omega_1 \left| e \left( t \right) \right| + \omega_2 e^2 \left( t \right) \right) dt + \omega_3 t_u$$

$$(15)$$

with

$$\omega_1 + \omega_2 = 1,$$

$$(16)$$

where $t_u$ is the rise time, $\omega_1$, $\omega_2$, and $\omega_3$ are weighting coefficients. In this work, the weighting coefficients are given as $\omega_1 = 0.99$, $\omega_2 = 0.01$, and $\omega_3 = 100$. The CPSO-ADRC structure for clamping force control system is shown in Figure 6.
5. Simulation and Comparison Results

In order to verify the effectiveness of CPSO-ADRC, PSO-ADRC is introduced as a comparison, which uses the PSO algorithm to optimize the ordinary ADRC and then simulation experiments are conducted. In Figure 7, the number of iterations is set to 100; it shows that the fitness function \( J \) of CPSO-ADRC can reach a lower value than that of PSO-ADRC during the iterative process. As shown in Table 4, the optimized key parameters have been obtained for both PSO-ADRC and CPSO-ADRC.

In Figure 8, step responses of ordinary ADRC, PSO-ADRC and CPSO-ADRC are drawn; it reveals that CPSO-ADRC has higher response speed and stronger robustness compared with ordinary ADRC and PSO-ADRC.

To further prove the superiority of CPSO-ADRC, the input signal is changed to square wave, which is \( \text{rin}(k) = \ldots \)
### Table 4: Optimized key parameters of ADRC.

<table>
<thead>
<tr>
<th>Controller</th>
<th>$\beta_{01}$</th>
<th>$\beta_{02}$</th>
<th>$\beta_{03}$</th>
<th>$\beta_1$</th>
<th>$\beta_2$</th>
</tr>
</thead>
<tbody>
<tr>
<td>PSO-ADRC</td>
<td>230.00</td>
<td>878.19</td>
<td>2076.66</td>
<td>5.90</td>
<td>0.010</td>
</tr>
<tr>
<td>CPSO-ADRC</td>
<td>229.96</td>
<td>792.86</td>
<td>2499.26</td>
<td>6.85</td>
<td>0.013</td>
</tr>
</tbody>
</table>

![Figure 7: Comparison between the iterative processes obtained from PSO-ADRC and CPSO-ADRC.](image1)

![Figure 8: Comparison between step responses obtained from ordinary ADRC, PSO-ADRC, and CPSO-ADRC.](image2)

The sign($\sin(kh)$) and the corresponding performances of three kinds of ADRC can be seen in Figures 9 and 10. It can be observed in Figure 9 that the CPSO-ADRC has better performance in responding speed, which is consistent with what Figure 8 implies. And it can also be observed from Figure 10 that CPSO-ADRC has excellent capability of command tracking.

In Figure 11, the control input performances of three controllers are compared. Ordinary ADRC produces significant chattering; PSO-ADRC also generates certain level...
of chattering, though much milder than ordinary ADRC; CPSO-ADRC performs the best in reducing chattering; no obvious chattering can be observed from its signal.

6. Conclusion

This study focused on improving the performance of active disturbance rejection controllers for clamping force control system via online optimization. The CPSO-ADRC combination was proposed aiming to achieve faster response and better command tracking performance. Simulation studies illustrated that the CPSO-ADRC in this work has exhibited better dynamic performance. Furthermore, the disturbance signal was designed with a Gaussian function to test the robustness of CPSO-ADRC and the result indicated that CPSO-ADRC has also showed better robustness. To sum all up, CPSO-ADRC has better control performance compared to ordinary ADRC and PSO-ADRC, with similar computational burden to the PSO-ADRC.

Conflicts of Interest

The authors declare that there are no conflicts of interest regarding the publication of this paper.
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Supplementary Materials

See supplementary material for the motion simulation video of the studied clamping mechanism. (Supplementary Materials)

References


