Research Article

Attractor and Vector Structure Analyses of Bursting Oscillation with Sliding Bifurcation in Filippov Systems

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The main purpose of this paper is to investigate the mechanism of sliding phenomenon in Filippov (nonsmooth) dynamical systems by attractor analysis and vector analysis. A corresponding simple model based on Chua’s circuit with periodic excitation was introduced as an example. The attractor analysis proposed in our previous work is used to discuss the complicated oscillations of the Filippov system. However, it failed to perfectly explain the sliding phenomena and establish an analytical method of constant voltage control. Therefore, the geometric structure and analytic conditions of sliding bifurcations in the general n-dimensional piecewise smooth system are discussed in detail by vector structure analysis. The prospects of practical application of this method are also discussed in the end.

1. Introduction

In practical engineering applications, there are many nonsmooth factors, such as switching [1] and impulse control [2]. The nonsmooth models describing the special dynamic behavior are generally divided into three categories: continuous nonsmooth systems [3], Filippov systems [4], and nonsmooth impulsive systems [5]. The Filippov-type (piecewise smooth) systems are characterized by discontinuous vector fields and continuous Jacobian matrices. The introduction of nonsmooth terms will result in a series of special dynamic behaviors near the interface, such as sliding, traversing, grazing, and jumping [6–9]. Due to the discontinuity of the vector field, the traditional attractor analysis method [10] employed to deal with smooth systems cannot be directly used to study the special dynamic behaviors occurring in nonsmooth region. Therefore, new theoretical methods are needed to analyze such problems.

The sliding motions [11, 12] with profound physical backgrounds exist widely in engineering practice. For example, in mechanical systems, the introduction of dry friction factor will result in the sliding phenomenon of mechanical oscillator [13]. Similarly, in the circuit system, sliding bifurcation indicates the conservation of voltage, which has certain guiding significance for the design of a constant voltage transformer. Therefore, the analytical conditions of sliding bifurcation [14] are of vital importance.

Taking Chua’s circuit system with bilateral diodes as an example, the bursting oscillation and sliding bifurcation are studied by equilibrium curve, bifurcation diagram, and phase portrait. The validity of the theoretical results is verified by calculation of the sliding time and analysis of simulation data near special points. The application prospects of methods proposed are discussed in the end.

2. Mathematical Model

The current-driven Chua’s circuit is shown in the Figure 1. The circuit consists of capacitors $C_1$ and $C_2$, inductor $L$, resistors $R$ and $R_N$, and current source $i_s$, in which $R_N$ is a nonlinear resistor. According to Kirchhoff’s current law and voltage law, the state equations of the circuit can be formed as
The volt-ampere characteristic function \( f(x) \) of the nonlinear resistance can be expressed in the form of cubic polynomial proposed by Hartley and Mossayebi [17] in the study of deformed Chua’s circuit as \( f(x) = mx + nx^3 \), where \( m = -8/7 \) and \( n = 2/7 \). It is noteworthy that the nonlinearity is weak due to the relatively small nonlinear coefficient.

Taking the simple Chua’s circuit system as an example, dynamic behaviors of Filippov systems with multiscale effect can be studied. By adding a bilateral diode as a nonlinear element, the dimensionless mathematical model of the modified circuit can be expressed as follows:

\[
\begin{align*}
\dot{x} &= \alpha (y - x - f(x)) + A \sin(\omega t), \\
\dot{y} &= x - y + z, \\
\dot{z} &= -\beta y.
\end{align*}
\]

where the \( x \)-value of external excitation is far less than the natural frequency, and when the frequency of external excitation is far less than the natural frequency \( \omega \), there will be two time scale effects. Nonsmooth interface \( x = 0 \) divides the original system into two subsystems located in different phase spaces.

### 3. Bifurcation Analysis of Two Subsystems

There are magnitude differences between external excitation frequency and natural frequency when \( 0 < \omega \ll 1 \). In any natural frequency period \( T_0 = 2\pi/\omega, t \in [t_0, t_0 + T_0] \), the external excitation will change between \( W_A = A \sin(\Omega t) \) and \( W_B = A \sin(\Omega t + (2\pi\Omega/\omega)) \). Because \( \Omega/\omega \ll 1 \), \( W_A \approx W_B \) which means that in any natural frequency period, the external excitation is almost a constant. Therefore, the whole external excitation term can be regarded as a slowly varying parameter and two generalized autonomous subsystems can be derived as follows.

When \( x < 0 \),

\[
S_1: \begin{cases} 
\dot{x} = \alpha (y - x - f(x)) + w - c, \\
\dot{y} = x - y + z, \\
\dot{z} = -\beta y.
\end{cases}
\]

When \( x > 0 \),

\[
S_2: \begin{cases} 
\dot{x} = \alpha (y + x^3 + bx) + w + c, \\
\dot{y} = x - y + z, \\
\dot{z} = -\beta y.
\end{cases}
\]

The nonsmooth interface \( \Sigma (\Sigma : (x, y, z) | x = 0) \) divides the phase space into two regions. The corresponding regions of \( x > 0 \) and \( x < 0 \) are represented as \( D_+ \) and \( D_- \), respectively. When the trajectory is located in \( D_+ \), it is restricted by subsystem \( S_1 \). Once the trajectory reaches \( D_+ \) through the
interface, the governing equation will be converted to $S_2$ immediately. The equilibrium point of two subsystems can be expressed as $E_0(x, y, z) = (X_0, 0, -X_0)$, where $X_0$ satisfying

$$a(x^3 + bX_0) + w + c \text{ sgn}(X_0) = 0. \quad (7)$$

The properties of the equilibrium point depend on the following characteristic equation:

$$\lambda^3 + \left(1 - ab - 3aaX_0^2\right)\lambda^2 + \left(\beta - a - ab - 3aaX_0^2\right) - a\beta(b + 3aX_0^2) = 0. \quad (8)$$

Therefore, the conditions for the stability of equilibrium point $E_0$ are as follows:

$$1 - ab - 3aaX_0^2 > 0,$$
$$a\beta(b + 3aX_0^2) < 0,$$
$$\left(1 - ab - 3aaX_0^2\right)(\beta - a - ab - 3aaX_0^2) + a\beta(b + 3aX_0^2) > 0. \quad (9)$$

Two types of codimension-one bifurcations can be observed, in which bifurcations occur under the following conditions:

$$\text{FB : } b + 3aX_0^2 = 0,$$
$$a(x^3 + bX_0) + w + c \text{ sgn}(X_0) = 0. \quad (10)$$

With $1 - ab - 3aaX_0^2 > 0,$ $\left(1 - ab - 3aaX_0^2\right)(\beta - a - ab - 3aaX_0^2) + a\beta(b + 3aX_0^2) > 0$.

Based on the Routh–Hurwitz stability criterion [2, 3], for a cubic polynomial,

$$P_3(z) = z^3 + a_1z^2 + a_2z + a_3,$$  
(11)

Its Routh–Hurwitz conditions can be formed as

$$a_1 > 0, \quad \begin{bmatrix} a_1 & 1 \\ a_3 & a_2 \end{bmatrix} > 0, \quad a_3(a_1a_2 - a_3) > 0, \quad \emptyset$$

$$a_1 > 0, \quad a_3 > 0, \quad a_1a_2 > a_3.$$  
(12)

When condition $a_1a_2 > a_3$ degenerates to $a_1a_3 = a_3$, a pair of pure imaginary roots appear in the solution of characteristic polynomial (8) indicating that Hopf bifurcation occurs. Therefore the conditions of Hopf bifurcation are as follows:

$$\text{HB : } a(x^3 + bX_0) + w + c \text{ sgn}(X_0) = 0.$$  
(13)

With $1 - ab - 3aaX_0^2 > 0,$ $a\beta(b + 3aX_0^2) < 0$.

The bifurcation and bursting oscillation behaviors of this system have been studied in our previous work [14]. Therefore, no further elaboration will be made here. The two-parameter bifurcation diagram and the equilibrium branches with bifurcations are shown in Figure 2 as a supplement to our previous work.

In Figure 2, FB$i_\pm$ and HB$i_\pm$ ($i = 1, 2$) corresponds to the different bifurcation points in different segments of equilibrium branches, respectively. The bifurcation lines divided the parametric plane into different regions marked by number $i$ ($i = 1, 2, \ldots, 7$), and the stability of the equilibrium points corresponding to the adjacent regions is opposite. The results of Figure 2 are consistent with those of our previous study.

4. Attractor Structure Analysis of Sliding Motion

It is noteworthy that all the bifurcation results obtained are related to slow-varying parameters [18–22]. Therefore, in order to reveal the complete bursting oscillation process and special dynamic behaviors of the system (4), the concept of transformed phase portrait (TPP) [23, 24] should be introduced.

4.1. TPP for the Trajectory $\Pi_{\Omega}$ : $\{x(t), y(t), z(t), t \in \mathbb{R}\}$. The traditional phase portrait reflects the relationship among state variables in the phase space $(x, y, z)$ or the projection planes $(x, y)$ or $(y, z)$. If the slowly varying parameter is regarded as a generalized state variable, the generalized phase trajectory can be obtained $\Pi_{\Omega_{GT}} : \{x(t), y(t), z(t), w(t), t \in \mathbb{R}\}$, where $w(t) = A \sin(\Omega t)$. The generalized phase trajectories in the generalized phase space $(x, y, z, w)$ and its projected phase plane are called transformed phase portraits, which are used to characterize the relationship between different state variables and slowly varying parameters.

Variable parameters $b$ were taken $b = 0.14$ and $b = 0.70$, respectively, while other parameters were fixed as

$$a = 6.9, \quad \beta = 10.0,$$
$$a = -\frac{2}{7}, \quad c = -2, \quad A = 6.0. \quad (14)$$

Therefore, $w$ varies periodically with frequency 0.01 in the interval $[-6.0, 6.0]$. Taking $A = 0$, the natural frequency of the system can be calculated. For $b = 0.14$, the equilibrium point of the two subsystems is approximately $E_i = (\pm 1.1662, 0.0, \mp 1.1662)$, with its corresponding eigenvalue as follows:

$$\lambda_1 = -7.920, \quad \lambda_2 = -0.0784 \pm 2.988i, \quad (15)$$

indicating that the natural frequency is about 2.988. Similarly, when $b = 0.70$, the coordinates of the three equilibrium points and their corresponding eigenvalues are...
\[ E_{1b} = (\pm 0.4517, 0.0, \mp 0.4517), \]
\[ \lambda_1 = 4.5159, \]
\[ \lambda_2 = -0.9463 \pm 2.6698i, \]
\[ E_{2b} = (\pm 1.2897, 0.0, \mp 1.2897), \]
\[ \lambda_1 = -6.0386, \]
\[ \lambda_2 = 0.01534 \pm 2.8797i, \]
\[ E_{3b} = (\pm 1.7414, 0.0, \mp 1.7417), \]
\[ \lambda_1 = -13.622, \]
\[ \lambda_2 = -0.2417 \pm 3.0923i. \]

Therefore, when the trajectory oscillates near different equilibrium points \( E_{ij} (j = 1, 2, 3) \), its natural frequencies are about 2.6698, 2.8797, and 3.0923.

4.2. Scenario 1: \( b = 0.14 \). The 3D phase portrait and bifurcation diagram with parameter \( b \) are shown in Figure 3. It can be seen that the attractor structure keeps the symmetry of the original system. The alternation of large-amplitude spiking states and small-amplitude quiescent states can be observed. Specially, the sliding phenomenon appears on the nonsmooth boundary indicating that the voltage of the capacitor remained a constant during this period of time. It should be pointed out that the phenomenon in Figure 3(b) does not mean the changes of periodicity (similar to period doubling bifurcation) but that the position of equilibrium points migrates due to the occurrence of sliding bifurcation which will be discussed in detail.

In order to reveal the mechanism of bursting oscillation, the overlap of the transformed phase portrait [14] with the equilibrium branches can be observed in Figure 4. \( EB_i^{(i)} (i = 1, 2; j = 1, 2, \ldots, 6) \) corresponds to the equilibrium curve segments of subsystem \( S_j(i = 1, 2) \), respectively. The solid part of the equilibrium line represents the stable equilibrium state, and the dashed line represents the unstable state. The black lines represent the equilibrium points that can be found in the corresponding region, while the red parts denote the equilibrium state that exists but is not located in the phase space of the corresponding system in control.

4.2.1. Bursting Oscillation Mechanism. As shown in Figure 4, the trajectory dominated by subsystem \( S_1 \) starts from point \( A_1 \) (\( w = -6.0 \)) and runs almost strictly along its stable equilibrium state \( EB_1^{(1)} \). Despite the existence of a stable limit cycle \( LC_1^{(1)} \) between the two supercritical Hopf bifurcation points \( HB_1^{(1)} \) and \( HB_2^{(1)} \), the trajectory still runs along the stable equilibrium, showing the quiescent state. The stability of limit cycles produced by these two Hopf bifurcations is opposite. They disappear after contact suggesting the occurrence of LPC bifurcation. When trajectory reaches the fold bifurcation point \( FB_1^{(1)} \), it jumps directly to the stable equilibrium. As time \( t \) continues to increase, the trajectory is attracted by the only stable equilibrium segment \( EB_6^{(2)} \) of subsystem \( S_2 \) until it intersects at \( A_2 \) with the nonsmooth boundary \( \Sigma \).

4.2.2. Supplementary Notes. It should be pointed out that although there are Hopf bifurcation points \( HB_1^{(1)} \) and \( HB_2^{(1)} \) on the equilibrium as shown in Figure 4, the trajectory still runs along the equilibrium due to the slow path effect which causes the bifurcation lag [25]. The phenomenon that the trajectory contacts the nonsmooth boundary and runs along \( \Sigma \) is called sliding motion. The sliding process is still in quiescent state until it reaches point \( A_3 \). Besides, it can be clearly seen from Figure 3(a) that when trajectory is located on the nonsmooth boundary (sliding motions), the original system will degenerate into a two-dimensional system:

\[
\begin{align*}
\dot{y} &= x - y + z, \\
\dot{z} &= -\beta y,
\end{align*}
\]

indicating that there will be a new equilibrium point (stable focus equilibrium point) on the interface \( x = 0 \), named...
pseudoequilibrium point. With the variation of parameter $\beta = R^2C_2/L$, the pseudoequilibrium will also bifurcate.

4.2.3. Attractor Structure When the Sliding Phenomenon Occurs. When the phase trajectory approaches the boundary from side $D_-$, it is attracted by $EB^{(1)}_6$ located in $D_+$ on subsystem $S_1$, so it has the tendency to move across the boundary to $D_+$. However, once the trajectory reaches $D_+$, the control system will be switched to $S_2$ immediately. The corresponding stable equilibrium $EB^{(2)}_6$ is located at $D_-$, which makes the trajectory return to $D_-$. The alternating control of two subsystems is the intrinsic cause of the sliding phenomenon.

4.2.4. Sliding Time Calculation. In our previous work [14], the sliding time has been estimated by the time required from $FB^{(1)}_1$ to $A_3$ with insufficient accuracy. In addition, the estimation method is not universal. New method are ought to be proposed, which will be discussed in the next section.

4.2.5. Discussion on Physical Meaning. In general, fold, Hopf, and sliding bifurcation, respectively, correspond to mutation, oscillation, and conservation of circuit component voltage in the circuit system. A constant voltage transformer is widely used in engineering practice. The above results show that constant voltage control can be achieved by choosing appropriate nonlinear resistors and the output time of constant voltage can accurately be controlled by the calculation method of sliding time. The key point is how to deduce the analytical expression of sliding motions (constant voltage) which will be discussed in Section 5.

4.3. Scenario 2: $b = 0.70$. In this case, a problem which was not well resolved in [14] should be pointed out. The overlap of the transformed phase portrait with the equilibrium branches is shown in Figure 5(a), and the local enlargement of time history is shown in Figure 5(b).

Although the distribution of attractor structure in case 2 is different from that in case 1, most of its dynamic behavior can be explained on the basis of previous studies, so it is not necessary to elaborate on this for article length consideration. It is noteworthy that as shown in local enlargement at the lower right corner of Figure 5(a), when the trajectory slides from point $A_2$, a small jump occurs immediately and then returns to the interface to continue sliding. This seemingly insignificant phenomenon cannot be well explained by attractor structure analysis due to the coexistence of multiple equilibrium points. In order to further study the special dynamic behavior of nonsmooth systems at nonsmooth boundary, it is obviously not enough to use attractor structure analysis alone. Therefore, based on differential inclusion theory and focusing on the local dynamics near the interface, another method to analyze such problems named vector structure analysis is proposed.
5. Structural Characteristics of Vector Field for Sliding Bifurcation

Consider a general $n$-dimensional piecewise smooth system:

$$
\dot{Y} = \begin{cases} 
F_1(Y), & h(Y) > 0, \\
F_2(Y), & h(Y) < 0.
\end{cases}
$$

(18)

The nonsmooth boundary can be defined as $\Sigma = \{Y \in R^{n+1} | h(Y) = 0\}$. The vector field of the sliding region can be represented by convex closures of $F_1(Y)$ and $F_2(Y)$:

$$
F_s(Y) = \frac{F_1(Y) + F_2(Y)}{2} + \frac{F_1(Y) - F_2(Y)}{2} \alpha(Y),
$$

(19)

where $-1 \leq \alpha(Y) \leq 1$. Assuming that $h_Y = (\partial h / \partial y_1, \partial h / \partial y_2, \ldots)^T = \nabla h$, so $h_Y F_s(Y) = 0$ (normal vector $h_Y$ is perpendicular to the nonsmooth interface, while $h_Y F_s(Y) = 0$ is parallel to the interface). Therefore, $\alpha(Y)$ can be expressed as

$$
\alpha(Y) = -\frac{h_Y (F_1 + F_2)}{h_Y (F_2 - F_1)}.
$$

(20)

The sliding region and its boundary can be expressed as

$$
\Sigma_s = \{Y \in \Sigma | -1 \leq \alpha(Y) \leq 1\},
$$

(21)

$$
\partial \Sigma_s^\pm = \{Y \in \Sigma | \alpha(Y) = \pm 1\},
$$

(22)

where $\partial \Sigma_s^-$ is the tangent point of $F_1(Y)$ on nonsmooth boundary $\Sigma$. So, $h_Y F_2(Y) = 0$ and $h_Y F_3(Y) = 0$. By deriving the two sides of formula (20) separately,

$$
\alpha_Y = -\frac{(h_Y (F_1 + F_2))_Y (h_Y (F_2 - F_1))}{(h_Y (F_2 - F_1)^2)}
$$

$$
+ \frac{(h_Y (F_1 + F_2))_Y (h_Y (F_2 - F_1))}{(h_Y (F_2 - F_1)^2)}
$$

$$
= -\frac{(h_Y F_2 Y + h_Y F_3 Y + h_Y F_4 Y + h_Y F_5 Y) (h_Y F_2 - h_Y F_1)}{(h_Y F_2 - h_Y F_1)^2}
$$

$$
+ \frac{(h_Y F_2 Y - h_Y F_3 Y) (h_Y F_2 - h_Y F_1)}{(h_Y F_2 - h_Y F_1)^2}
$$

(23)

$\partial \Sigma_s^-$ satisfying $F_s(Y) = F_1(Y)$, indicating that $h_Y F_1 = 0$. Combining with (23), normal vectors on $\partial \Sigma_s^-$ can be expressed as

$$
\alpha_Y = -\frac{2}{h_Y F_2} h_Y F_1 Y.
$$

(24)

The physical meaning of differential inclusion theory can be understood simply through dry friction: sliding friction corresponds to the boundary of sliding region, static friction corresponds to the interior of sliding region, and the numerical value of static friction depends on the equilibrium condition.
5.1. Geometric Classification of Sliding Bifurcations. According to the geometric structure of the periodic solution trajectory intersecting with the sliding region and its boundary, the sliding bifurcation can be roughly divided into the following four categories, as shown in Figure 6.

The characteristic of crossing sliding bifurcation is that the trajectory crosses the boundary of the sliding region at the bifurcation point and leaves (as shown in Figure 6(a) trajectory b). Then the sliding phenomena occur when the contact points enter the sliding area with the change of parameters (as shown in the trajectory c of Figure 6(a)).

The grazing sliding bifurcation, as the name implies, grazes along the boundary of the sliding region at the bifurcation point and leaves (as shown in Figure 6(b) track b). Then, the sliding phenomena occur when the contact points enter the sliding area with the change of parameters (as shown in the trajectory c of Figure 6(b)).

The switching sliding bifurcation is similar to crossing sliding bifurcation, but the difference is that a section of track is still in the sliding region after traversing the boundary of the sliding region at the bifurcation point (as shown in Figure 6(c) trajectory b). Then, a segment of the trajectory remains in the sliding region as the parameters change (as shown in Figure 6(c) trajectory c).

Unlike the other three bifurcations, the bifurcated trajectories of the adding sliding bifurcation are completely located in the sliding region and tangent to the boundary of the sliding region at the bifurcation point (as shown in track b of Figure 6(d)). As the parameters change, a section of the track is still located in the sliding region (as shown in track c of Figure 6(d)).

5.2. Analytical Conditions for Sliding Bifurcation. The above four kinds of sliding bifurcations can accurately be described by analytic conditions.

5.2.1. Crossing Sliding Bifurcation and Grazing Sliding Bifurcation. Because the bifurcation point \( Y^* \) is located on the nonsmooth boundary,

\[
h(Y^*) = 0, \\ h_Y(Y^*) \neq 0. \tag{25}
\]

Due to \( Y^* \in \partial \Sigma^- \),

\[
\frac{d}{dt} h(\phi(Y^*,t)) \bigg|_{t=0} = h_Y F_1(Y^*) = 0, \tag{26}
\]

where \( \phi \) is the corresponding smooth solution manifold. In addition, after crossing the interface, the trajectory moves away from the sliding area, so

\[
\frac{d}{dt} a(\phi(Y^*,t)) \bigg|_{t=0} = a_Y F_1(Y^*) < 0. \tag{27}
\]

By the law of derivation of the dot product and the combination of formulas (24), (26), and (27),

\[
\frac{d^2}{dt^2} h(\phi(Y^*,t)) \bigg|_{t=0} = h_Y F_{1Y} F_1(Y^*) > 0. \tag{28}
\]

5.2.2. Switching Sliding Bifurcation. This situation also satisfies the two basic conditions (25) and (26), but the track crosses the sliding area at the bifurcation point and remains in the sliding area; therefore,

\[
\frac{d}{dt} a(\phi(Y^*,t)) \bigg|_{t=0} = a_Y F_1(Y^*) > 0, \tag{29}
\]

\[
\frac{d^2}{dt^2} h(\phi(Y^*,t)) \bigg|_{t=0} = h_Y F_{1Y} F_1(Y^*) < 0. \tag{30}
\]

5.2.3. Adding Sliding Bifurcation. The two basic conditions (25) and (26) are still valid, but in this case, the bifurcation trajectory is completely located within the sliding region and tangent to the boundary of the sliding region at the bifurcation point, so the conditions (28) and (30) for the above three cases degenerate:

\[
\frac{d^2}{dt^2} h(\phi(Y^*,t)) \bigg|_{t=0} = h_Y F_{1Y} F_1(Y^*) = 0. \tag{31}
\]

Because the bifurcation trajectory has a minimum value at \( \partial \Sigma^- \), there is a nondegeneration condition:

\[
\frac{d^2}{dt^2} a(\phi(Y^*,t)) \bigg|_{t=0} = a_Y F_{1Y} F_1(Y^*) > 0. \tag{32}
\]

By the law of derivation of the dot product and the combination of formulas (24), (26), and (32),

\[
\frac{d^3}{dt^3} h(\phi(Y^*,t)) \bigg|_{t=0} = h_Y (F_{1Y})^2 F_1(Y^*) < 0. \tag{33}
\]

5.3. Vector Field Structure Characteristics of Sliding Bifurcation. Based on the previous analysis, the vector field structure of sliding bifurcation can be summarized.

Regardless of the type of sliding bifurcation, as long as the trajectory is located in the sliding region, its vector field must have the structural characteristics as shown in Figure 7(a).

Sliding phenomena occur when there are opposite vector fields on both sides of the nonsmooth interface. The direct reason is that the trajectory is forced by the opposite vector field near the boundary and thus can only oscillate along the boundary. When the limit of step size approaches zero, it is a strict sliding motion. With opposite vector field in Figure 7(a), if trajectory locates in \( D_- \), it will be pushed to nonsmooth boundary. For the same reason, if it locates in \( D_+ \), it will also approach the boundary. Therefore, it can be understood as that the trajectory is squeezed by the vector field and can only strictly slide along the nonsmooth boundary. This result is consistent with the conclusions of attractor structure analysis. In contrast, when vector fields with the same sign exist on both sides of the nonsmooth boundary, the trajectory traverses the interface directly, as shown in Figure 7(b).

In scenario 2, the transient jumping phenomenon after sliding is caused by the change of external excitation in the sliding region, which leads to the change of vector field, until the vector field on one side weakens gradually and eventually reverses. It should be pointed out that although there is only one stable attractor on both sides of the interface, the
complete sliding phenomenon may not occur. This is because
the stability of the attractor does not directly determine the
direction of the normal component of the vector field at the
interface, as shown in diagrammatic sketch Figure 8.

For instance, supposing that subsystem defined in $D_+$
has a stable focus (or limit cycle) $P_+$ in $D_-$. The direction of
the vector field in its domain of attraction is shown by the
blue arrows. It can be clearly seen that although $P_+$ is attrac-
tive to the phase trajectory in $D_+$ in the long run, its
vector field on the side of $D_+$ does not point to the non-
smooth boundary (as shown in the dotted part). Therefore,
although the attractor structure analysis methods (such as
fast-slow analysis method) have some advantages in
studying the global dynamic behaviors of nonsmooth peri-
dodic excitation system, it also has some limitations in
dealing with some special local dynamic details such as
sliding bifurcation.

5.4. Numerical Verification. In order to better understand
the mechanism of sliding bifurcation, the sliding process in
case 2 is numerically analyzed as Table 1.

It is noteworthy that in sliding region $A_2 \rightarrow A_3$, the
signs of vector field $\dot{x}$ on both sides of nonsmooth boundary
$x = 0$ are opposite and the difference is exactly equal to $2|c| =
2 \times 2.0 = 4.0$ ($c$ is the coefficient of the symbolic function
shown in formula (14)). Based on the analytical expressions
(20)–(22) of the sliding region of the upper section, it can be
found that the numerical results are in good agreement with
the theoretical results.

5.5. Discussion on Physical Meaning. The proposed analytical
conditions of sliding bifurcation imply that the constant
voltage can be controlled by changing the external excitation
(AC source) rather than the properties of the circuit ele-
ments themselves. Besides, the sliding phenomenon also
exists in mechanical systems such as spring oscillator
considering dry friction factor, etc. Actually, in engineering
practice, the sliding (sticking) phenomenon is not always
expected. For instance, the occurrence of sliding phenom-
emon of the lathes for producing models with spring os-
cillator means the production of defective products.
Therefore, finding out a way to eliminate the sliding motions

Figure 6: (a) Crossing, (b) grazing, (c) switching, and (d) adding sliding bifurcations.

Figure 7: The structure of the vector field in (a) sliding region and (b) traversing region.
is also of great concern. Controlling the frequency of external excitation instead of changing the property of the material itself is a relatively simple and effective solution because the change of external excitation invalidates the necessary condition of sliding bifurcation mentioned in the manuscript. This part of the content will be further explored in our next work.

6. Conclusions

The introduction of bidirectional diodes in the circuit system will result in the occurrence of sliding bifurcation indicating the conservation of voltage. The selection of different nonlinear resistors corresponds to different sliding time and modes. Sliding motions can be divided into four categories according to the corresponding geometric characteristics. The specific expression of sliding area and its boundary along with the special structure of vector field are presented. Vector fields on both sides of nonsmooth boundary are opposite in sliding region while remain the same in traversing region.

In addition, this paper mainly focuses on the single variable of nonlinear resistance. Coupling effects between multiple factors will be discussed separately in other literature.

Data Availability

All data generated or analyzed during this study are included in this published article. Requests for material should be made to the corresponding author.

Conflicts of Interest

The authors declare that there are no conflicts of interest regarding the publication of this article.

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