

Research Article

Dynamic Model of a Wind Turbine for the Electric Energy Generation

José de Jesús Rubio,¹ Luis Arturo Soriano,² and Wen Yu²

¹ *Sección de Estudios de Posgrado e Investigación, ESIME Azcapotzalco, Instituto Politécnico Nacional, Avenida de las Granjas No. 682, Colonia Santa Catarina, 02250 México, DF, Mexico*

² *Departamento de Control Automático, CINVESTAV, Instituto Politécnico Nacional, 07360 México, DF, Mexico*

Correspondence should be addressed to José de Jesús Rubio; jrubioa@ipn.mx

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A novel dynamic model is introduced for the modeling of the wind turbine behavior. The objective of the wind turbine is the electric energy generation. The analytic model has the characteristic that considers a rotatory tower. Experiments show the validity of the proposed method.

1. Introduction

Researchers are often trying to improve the total power of a wind turbine. The dynamic model of a wind turbine plays an important role in some applications as the control, classification, or prediction.

Some authors have proposed the equations to model the dynamic behavior of the wind turbine as shown in [1–5].

This paper presents a novel dynamic model of a wind turbine; the first dynamic model is for the wind turbine, the second is for the tower, and both are related. Because the rotatory tower can turn, it may help the wind turbine to increase the air intake. Some companies propose wind turbines that consider rotatory blades; however, in this study, a rotatory tower is proposed instead of the rotatory blades because the first dynamic model is easier to obtain than the second.

The proposed model is linear in the states as the works analyzed by [6–10] show. The technique provides an acceptable approximation of the wind turbine behavior.

The paper is structured as follows. In Section 2, the dynamic model of a windward wind turbine of three blades with a rotatory tower is introduced. In Section 3, the simulations of the dynamic model are compared with the real

data obtained by a real wind turbine prototype. Finally, in Section 4, the conclusion and future research are detailed.

2. Dynamic Model of the Wind Turbine with a Rotatory Tower

This section is divided into four parts: the first is the description of the mechanic model, the second is the description of the aerodynamic model, the third is the description of the electric model, and, finally, the fourth is the combination of the aforementioned models to obtain the final dynamic model.

The dynamic model of the wind turbine is, first, the equations that represent the change between the wind energy and mechanic energy and, second, the equations that represent the change between the mechanic energy and electric energy.

2.1. The Mechanic Model. A windward wind turbine of three blades with a rotatory tower is considered. First, the Euler Lagrangian method [11, 12] is used to obtain the model that represents the change from the wind energy to mechanic energy for the wind turbine and the change from the electric energy to mechanic energy for the tower. The masses are

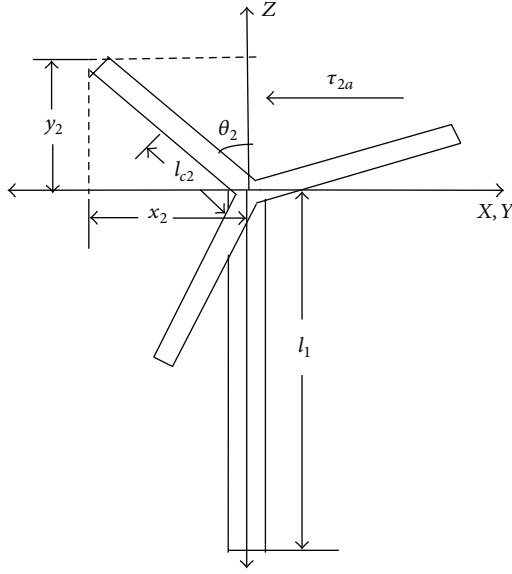


FIGURE 1: Lateral view of the wind turbine.

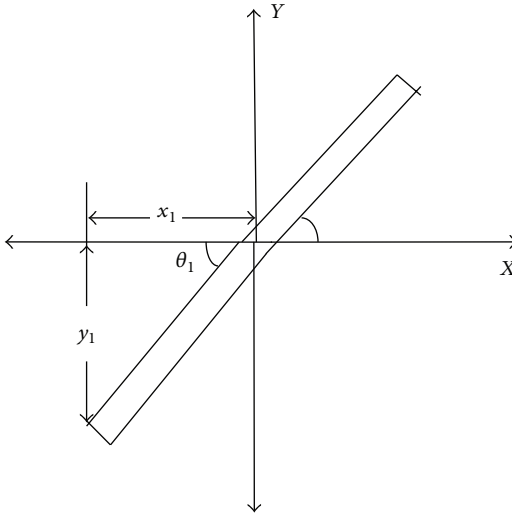


FIGURE 2: Upper view of the wind turbine.

concentrated at the center of mass. Consider the lateral view of Figure 1 and upper view of Figure 2.

From Figures 1 and 2, it can be seen that

$$\begin{aligned} x_2 &= -l_{c2}S_2(-C_1) = l_{c2}S_2C_1, \\ y_2 &= -l_{c2}S_2(-S_1) = l_{c2}S_2S_1, \end{aligned} \quad (1)$$

where $S_1 = \sin(\theta_1)$, $S_2 = \sin(\theta_2)$, $C_1 = \cos(\theta_1)$, $C_2 = \cos(\theta_2)$, θ_1 is the angular position of the tower motor in rad, θ_2 is the angular position of a wind turbine blade in rad, and l_{c2} is the length to the center of the wind turbine blade in m.

Consequently, the kinetic energy K_1 , K_2 and potential energy P_1 , P_2 are given as

$$\begin{aligned} K_1 &= 0, \\ K_2 &= \frac{1}{2}m_2l_{c2}^2S_2^2\dot{\theta}_1^2 + \frac{1}{2}m_2l_{c2}^2\dot{\theta}_2^2, \\ P_1 &= m_1gl_{c1}, \\ P_2 &= l_1 + m_2gl_{c2}C_2, \end{aligned} \quad (2)$$

where m_1 is the tower mass in kg, m_2 is the blade mass in kg, g is the acceleration gravity in m/s^2 , l_1 is the constant length of the tower in m, and l_{c1} is the length to the center of the tower in m. The torques τ_{T1a} and τ_{T2a} are given as

$$\begin{aligned} \tau_{T1a} &= \tau_{1a} - k_{b1}\theta_1 - b_{b1}\dot{\theta}_1, \\ \tau_{T2a} &= \tau_{2a} - k_{b2}\theta_2 - b_{b2}\dot{\theta}_2, \end{aligned} \quad (3)$$

where τ_{2a} is the torque of the generator moved by the blade in $\text{kg m}^2 \text{rad/s}^2$, τ_{1a} is the torque of the motor used to move the tower in $\text{kg m}^2 \text{rad/s}^2$, k_{b1} and k_{b2} are the spring effects presented when the blade is near to a stop in $\text{kg m}^2/\text{s}^2$, and b_{b1} and b_{b2} are the shock absorbers in $\text{kg m}^2 \text{rad/s}$. Using the Euler Lagrange method in [11, 12] gives the following equations:

$$\begin{aligned} m_2l_{c2}^2S_2^2\ddot{\theta}_1 + 2m_2l_{c2}^2S_2C_2\dot{\theta}_1\dot{\theta}_2 + b_{b1}\dot{\theta}_1 + k_{b1}\theta_1 &= \tau_{1a}, \\ m_2l_{c2}^2\ddot{\theta}_2 + m_2gl_{c2}S_2 + b_{b2}\dot{\theta}_2 + k_{b2}\theta_2 &= \tau_{2a}. \end{aligned} \quad (4)$$

The angular position of the blade 1 is related to the angular position of the blades 2 and 3 as follows:

$$\begin{aligned} \theta_3 &= \theta_2 + \frac{2}{3}\pi \text{ rad}, \\ \theta_4 &= \theta_2 + \frac{4}{3}\pi \text{ rad}, \end{aligned} \quad (5)$$

where θ_3 and θ_4 are the angular positions for the blades 2 and 3, respectively. Then, using (4), it yields the equations for blades 2 and 3 as a function of θ_2 as follows:

$$\begin{aligned} m_2l_{c2}^2S_{2+(2/3)\pi}^2\ddot{\theta}_1 + 2m_2l_{c2}^2S_{2+(2/3)\pi}C_{2+(2/3)\pi}\dot{\theta}_1\dot{\theta}_2 \\ + b_{b1}\dot{\theta}_1 + k_{b1}\theta_1 &= \tau_{1b}, \\ m_2l_{c2}^2\ddot{\theta}_2 + m_2gl_{c2}S_{2+(2/3)\pi} + b_{b2}\dot{\theta}_2 + k_{b2}\left(\theta_2 + \frac{2}{3}\pi\right) &= \tau_{2b}, \\ m_2l_{c2}^2S_{2+(4/3)\pi}^2\ddot{\theta}_1 + 2m_2l_{c2}^2S_{2+(4/3)\pi}C_{2+(4/3)\pi}\dot{\theta}_1\dot{\theta}_2 \\ + b_{b1}\dot{\theta}_1 + k_{b1}\theta_1 &= \tau_{1c}, \\ m_2l_{c2}^2\ddot{\theta}_2 + m_2gl_{c2}S_{2+(4/3)\pi} + b_{b2}\dot{\theta}_2 + k_{b2}\left(\theta_2 + \frac{4}{3}\pi\right) &= \tau_{2c}, \end{aligned} \quad (6)$$

where τ_{1b} , τ_{2b} , τ_{1c} , and τ_{2c} are the torques applied to move the blades 2 and 3, respectively. Adding $S_2^2 + S_{2+(2/3)\pi}^2 + S_{2+(4/3)\pi}^2$,

$S_2 C_2 + S_{2+(2/3)\pi} C_{2+(2/3)\pi} + S_{2+(4/3)\pi} C_{2+(4/3)\pi}$, and $S_2 + S_{2+(2/3)\pi} + S_{2+(4/3)\pi}$, respectively, it gives

$$S_2^2 + S_{2+(2/3)\pi}^2 + S_{2+(4/3)\pi}^2 = 1.5, \quad (8)$$

$$S_2 C_2 + S_{2+(2/3)\pi} C_{2+(2/3)\pi} + S_{2+(4/3)\pi} C_{2+(4/3)\pi} = 0, \quad (9)$$

$$S_2 + S_{2+(2/3)\pi} + S_{2+(4/3)\pi} = 0. \quad (10)$$

Now, adding the three equations of (4), (6), and (7) and using (8), (9), and (10), it gives

$$4.5m_2 l_{c2}^2 \ddot{\theta}_1 + 3b_{b1} \dot{\theta}_1 + 3k_{b1} \theta_1 = \tau_1, \quad (11)$$

$$3m_2 l_{c2}^2 \ddot{\theta}_2 + 3b_{b2} \dot{\theta}_2 + 3k_{b2} \theta_2 + 2\pi k_{b2} = \tau_2,$$

where $\tau_1 = \tau_{1a} + \tau_{1b} + \tau_{1c}$ and $\tau_2 = \tau_{2a} + \tau_{2b} + \tau_{2c}$. From Figures 1 and 2, τ_2 of (11) is defined as follows:

$$\begin{aligned} \tau_2 &= C_1 F_2, \\ F_2 &= F_{2a} + F_{2b} + F_{2c}, \end{aligned} \quad (12)$$

where $C_1 = \cos(\theta_1)F_{2a}$ and F_{2b} and F_{2c} are the force of the air received by the three blades. Equation (12) describes the assumption that the air goes in one direction; if $\theta_1 = 0$, then the maximum air intake moves the blades of the wind turbine, but if the tower turns to the left or to the right and θ_1 changes, then the wind turbine turns and the air intake decreases. From Figures 1 and 2 and [12], τ_1 of (11) is defined as follows:

$$\tau_1 = k_m i_1, \quad (13)$$

where k_m is a motor magnetic flux constant of the tower in Wb and i_1 is the motor armature current of the tower in A. Equations (11), (12), and (13) are the main equations of the mechanic model that represents the wind turbine and tower.

2.2. Aerodynamic Model. The aerodynamic model is the dynamic model of the torque applied to the blades. The mechanic power captured by the wind turbine P_a is given by [2–5]

$$P_a = F_{2a} \dot{\theta}_2 = \frac{1}{2} \rho A C_p(\lambda, \beta) V_\omega^3, \quad (14)$$

where ρ is the air density in Kg/m^3 , $A = \pi R^2$ is the area swept by the rotor blades in m^2 with radius R in m, V_ω is the wind speed in m/s, and $C_p(\lambda, \beta)$ is the performance coefficient of the wind turbine, whose value is a function of the tip speed ratio λ , defined as [2–5]

$$\lambda = \frac{\dot{\theta}_2 R}{V_\omega}. \quad (15)$$

For the purpose of simulation, the following model of $C_p(\lambda, \beta)$ is presented [2–5]:

$$C_p(\lambda, \beta) = c_1 \left(\frac{c_2}{\lambda_i} - c_3 \beta - c_4 \right) e^{-c_5/\lambda_i} + c_6 \lambda, \quad (16)$$

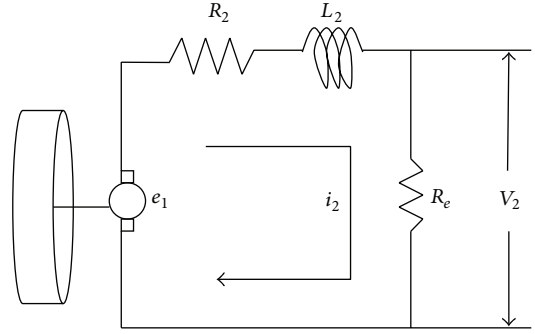


FIGURE 3: The wind turbine generator.

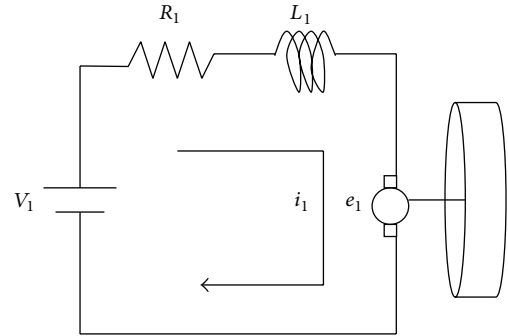


FIGURE 4: The tower motor.

where

$$\frac{1}{\lambda_i} = \frac{1}{\lambda + 0.08\beta} - \frac{0.035}{\beta^3 + 1} \quad (17)$$

and the coefficients are $c_1 = 0.5176$, $c_2 = 116$, $c_3 = 0.4$, $c_4 = 5$, $c_5 = 21$, $c_6 = 0.0068$, and β is the blade pitch angle in rad. Using (12), (14), (15), (16), and (17) gives the dynamic model of the torque applied to the wind turbine blades as follows:

$$\begin{aligned} \tau_2 &= C_1 (F_{2a} + F_{2b} + F_{2c}), \\ F_{2a} &= \frac{1}{2\dot{\theta}_2} \rho A C_p(\lambda, \beta) V_\omega^3, \\ F_{2b} &= \frac{1}{2\dot{\theta}_2} \rho A C_p(\lambda, \beta) V_\omega^3, \\ F_{2c} &= \frac{1}{2\dot{\theta}_2} \rho A C_p(\lambda, \beta) V_\omega^3, \end{aligned} \quad (18)$$

where C_1 is defined in (12), λ is defined in (15), $C_p(\lambda, \beta)$ is defined in (16), and λ_i is defined in (17).

2.3. The Electric Model. Now, analyze the change from the mechanic to electric energy for the wind turbine generator and the electric energy to mechanic energy for the tower motor. Figure 3 shows the wind turbine generator and Figure 4 shows the tower motor.

In [12, 13], they presented the model of a motor, and similarly, from Figure 3, a model for the generator is obtained

by using the Kirchhoff voltage law. Therefore, the dynamic models of the motor and generator are as follows:

$$\begin{aligned} V_1 &= R_1 i_1 + L_1 \dot{i}_1 + k_1 \dot{\theta}_1, \\ k_2 \dot{\theta}_2 &= R_2 i_2 + L_2 \dot{i}_2 + V_2, \end{aligned} \quad (19)$$

where k_1 is the motor back emf constant in V s/rad, k_2 is the generator back emf constant in V s/rad, R_1 is the motor armature resistance in Ω , R_2 is the generator armature resistance in Ω , L_1 is the motor armature inductance in H, L_2 is the generator armature inductance in H, V_1 is the motor armature voltage in V, V_2 is the generator armature voltage in V, i_1 is the motor armature current in A, and i_2 is the generator armature current in A. For the generator of this paper, $V_2 = R_e i_2$. Thus, (19) becomes

$$\begin{aligned} V_1 &= R_1 i_1 + L_1 \dot{i}_1 + k_1 \dot{\theta}_1, \\ k_2 \dot{\theta}_2 &= (R_2 + R_e) i_2 + L_2 \dot{i}_2, \\ V_2 &= R_e i_2. \end{aligned} \quad (20)$$

2.4. The Final Dynamic Model. Thus, (11), (13), and (18) that represent the change between the wind energy and mechanic energy and (20) that represents the change between the mechanic energy and electric energy are considered as the wind turbine dynamic model with a rotatory tower.

Define the state variables as $x_1 = i_2$, $x_2 = \theta_2$, $x_3 = \dot{\theta}_2$, $x_4 = i_1$, $x_5 = \theta_1$, $x_6 = \dot{\theta}_1$, the inputs as $u_1 = F_2$ and $u_2 = V_1$, and the output as $y = V_2$. Consequently, the dynamic model of (11), (13), (15), (18), and (20) becomes

$$\begin{aligned} \dot{x}_1 &= -\frac{(R_2 + R_e)}{L_2} x_1 + \frac{k_2}{L_2} x_3, \\ \dot{x}_2 &= x_3, \\ \dot{x}_3 &= -\frac{k_{b2}}{m_2 l_{c2}^2} x_2 - \frac{b_{b2}}{m_2 l_{c2}^2} x_3 - \frac{2\pi k_{b2}}{3m_2 l_{c2}^2} + \frac{\cos(x_5)}{3m_2 l_{c2}^2} u_1, \\ \dot{x}_4 &= -\frac{R_1}{L_1} x_4 - \frac{k_1}{L_1} x_6 + \frac{1}{L_1} u_2, \\ \dot{x}_5 &= x_6, \\ \dot{x}_6 &= -\frac{3k_{b1}}{4.5m_2 l_{c2}^2} x_5 - \frac{3b_{b1}}{4.5m_1 l_{c2}^2} x_6 + \frac{k_m}{4.5m_2 l_{c2}^2} x_4, \\ y &= R_e x_1, \\ u_1 &= F_{2a} + F_{2b} + F_{2c}, \\ F_{2a} &= \frac{1}{2x_3} \rho A C_p(\lambda, \beta) V_\omega^3, \\ F_{2b} &= \frac{1}{2x_3} \rho A C_p(\lambda, \beta) V_\omega^3, \\ F_{2c} &= \frac{1}{2x_3} \rho A C_p(\lambda, \beta) V_\omega^3, \\ \lambda &= \frac{x_3 R}{V_\omega}, \end{aligned} \quad (21)$$

where $C_p(\lambda, \beta)$ is defined in (16) and λ_i is defined in (17).



FIGURE 5: Prototype of a wind turbine with rotatory tower.

TABLE 1: Parameters of the prototype.

Parameter	Value
l_{c2}	0.5 m
m_2	0.5 kg
k_{b2}	$1 \times 10^{-6} \text{ kgm}^2/\text{s}^2$
b_{b2}	$1 \times 10^{-1} \text{ kgm}^2/\text{rad/s}$
k_2	0.45 Vs/rad
R_2	6.96 Ω
L_2	$6.031 \times 10^{-1} \text{ H}$
R	$l_{c2} \text{ m}$
ρ	1.225 kg/m^3
g	9.81 m/s^2
R_e	30 Ω
k_m	0.09 Wb
k_{b1}	$1 \times 10^{-6} \text{ kgm}^2/\text{s}^2$
b_{b1}	$1 \times 10^{-1} \text{ kgm}^2/\text{rad/s}$
k_1	0.0045 Vs/rad
R_1	18 Ω
L_1	$6.031 \times 10^{-1} \text{ H}$
V_ω	5 m/s
β	0.5 rad

Remark 1. The model of this research considers a direct current motor and a direct current generator. This wind turbine is proposed to be used on the house roof for the daily usage to feed a light as a renewable source. If more than one wind turbine is used, more electricity is generated. The proposed model could be extended to other kinds of machines by changing (20).

Remark 2. Note that the dynamic model states of the wind turbine with a rotatory tower (21) is a linear system [6–10].

3. Experiments to Validate the Dynamic Model

Figure 5 shows the prototype of a wind turbine with a rotatory tower which is considered for the simulations of the dynamic model. This prototype has three blades with a rotatory tower which does not use a gear box. Table 1 shows the parameters of the prototype. The parameters m_2 and l_{c2} are obtained from

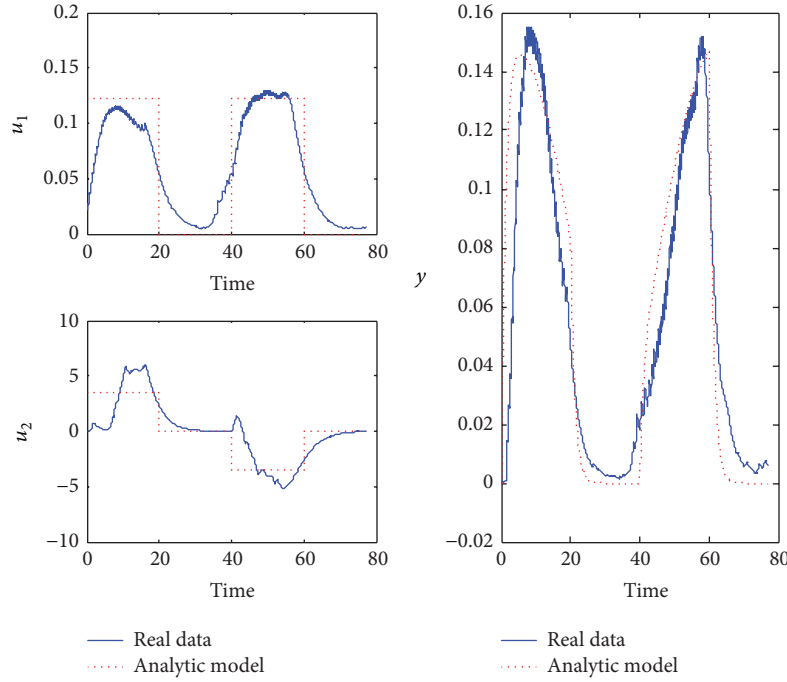


FIGURE 6: Inputs and output of the wind turbine of Example 1.

the wind turbine blades. The parameters R_1 , L_1 , and k_1 are obtained from the tower motor. The parameters k_2 , R_2 , R_e , and L_2 are obtained from the wind turbine generator. The parameters R , ρ , V_ω , and β are obtained from [2–5].

The dynamic model of the wind turbine with a rotatory tower is given by (21) with the parameters of Table 1. 1×10^{-5} are considered as the initial conditions for the plant states $x_1 = i_2$, $x_2 = \theta_2$, $x_3 = \dot{\theta}_2$, $x_4 = i_1$, $x_5 = \theta_1$, and $x_6 = \dot{\theta}_1$. The root mean square error (RMSE) is used, and it is given as [13–15]

$$\text{RMSE} = \left(\frac{1}{T} \int_0^T e^2 d\tau \right)^{1/2}, \quad (22)$$

where $e^2 = e_{uj}^2 = (u_{jr} - u_j)^2$, $e^2 = e_{xi}^2 = (x_{ir} - x_i)^2$, or $e^2 = e_y^2 = (y_r - y)^2$ and u_{jr} , x_{ir} , and y_r are the real data of u_j , x_i , and y , respectively, $i = 1, 2, \dots, 6$, $j = 1, 2$.

3.1. Example 1: The First Behavior. Figures 6 and 7 show the wind turbine inputs, outputs, and states. Table 2 shows the RMSE for the errors.

From Figures 6 and 7, it can be seen that the dynamic model has good behavior described as follows: (1) from 0 s to 2 s, both inputs are fed; consequently, the tower moves far of the maximum air intake, the generator current is decreased, and the wind turbine blades stop moving; (2) from 2 s to 4 s, both inputs are not fed; consequently, current is not generated and both the tower and wind turbine blades do not move; (3) from 4 s to 6 s, both inputs are fed, but the air intake is positive and tower voltage is negative; consequently, the tower returns to the maximum air intake, the generator current is increased, and the wind turbine blades move; (4) from 6 s to 8 s, both

TABLE 2: The RMSE for Example 1.

	RMSE
e_{u1}^2	0.0204
e_{u2}^2	1.1835
e_y^2	0.0171
e_{x1}^2	5.7158×10^{-4}
e_{x2}^2	0.9843
e_{x3}^2	0.0470
e_{x4}^2	0.0658
e_{x5}^2	0.0872
e_{x6}^2	0.0163

inputs are not fed; consequently, current is not generated and the tower and wind turbine blades do not move. The dynamic model is a good approximation of the wind turbine behavior because the signals of the first are near to the signals of the second. From Table 2, it is shown that the dynamic model is a good approximation of the real process because the RMSE is near to zero.

3.2. Example 2: The Second Behavior. Figures 8 and 9 show the wind turbine inputs, outputs, and states. Table 3 shows the RMSE for the errors.

From Figures 8 and 9, it can be seen that the dynamic model has good behavior described as follows: (1) from 0 s to 2 s, the input air is fed and the tower input is not fed; consequently, the tower remains in the maximum air intake, the generator current is maximum, and the wind turbine blades have motion; (2) from 2 s to 4 s, the air is not fed and

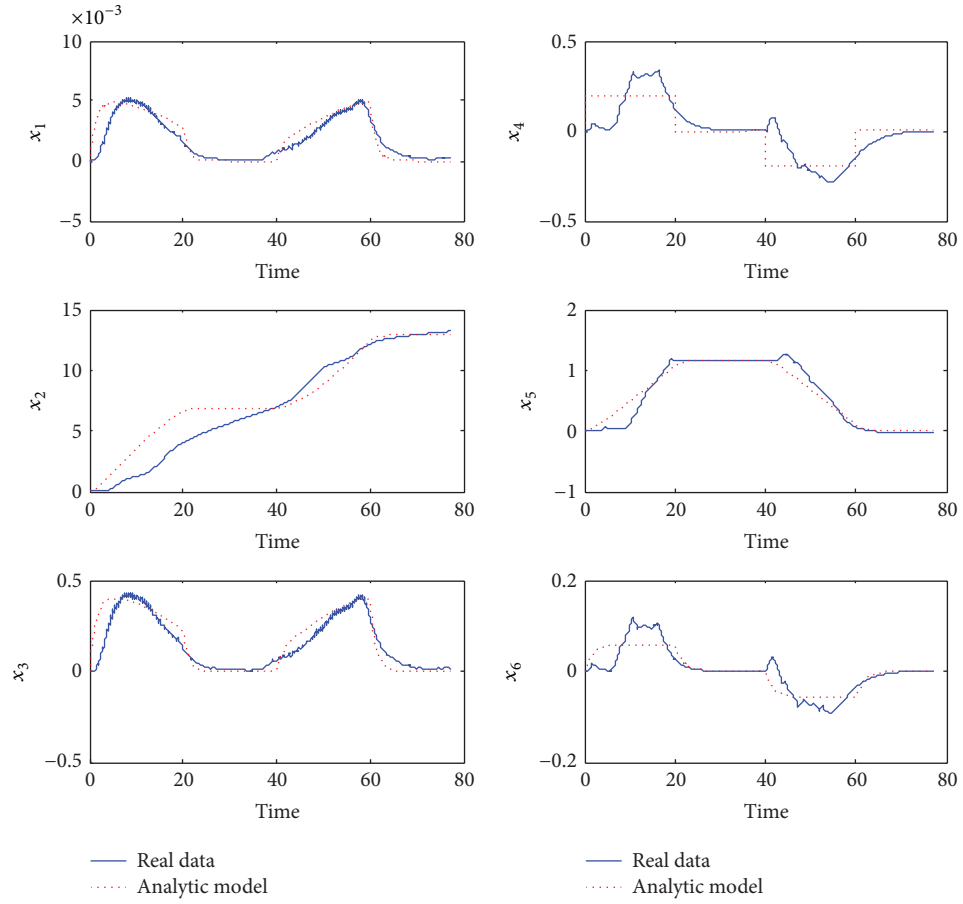


FIGURE 7: States of the wind turbine of Example 1.

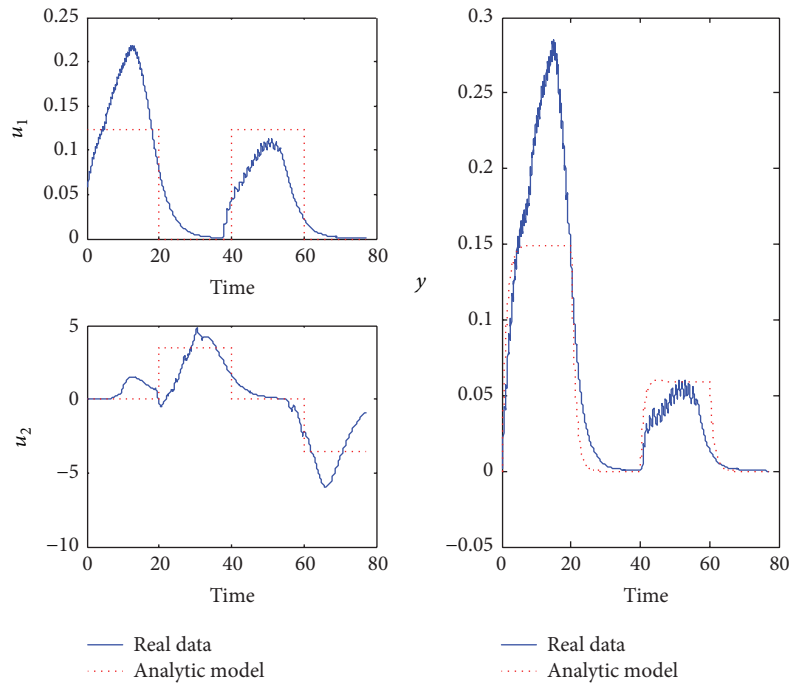


FIGURE 8: Inputs and output of the wind turbine of Example 2.

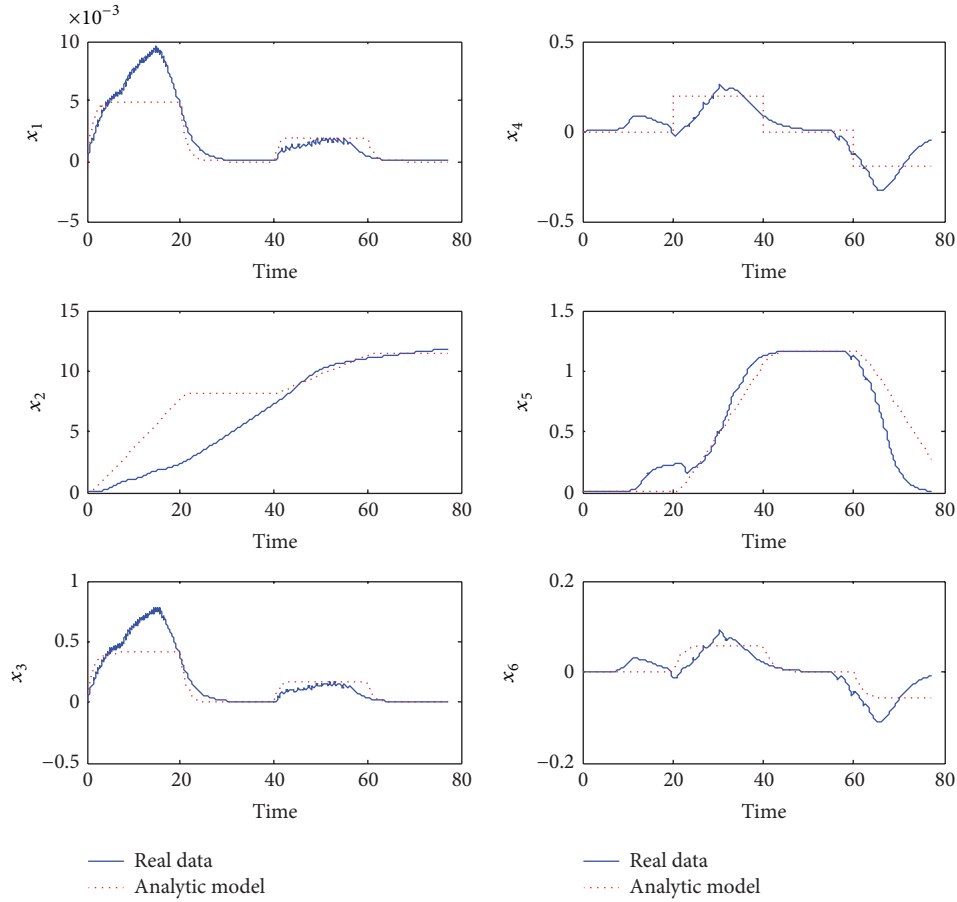


FIGURE 9: States of the wind turbine of Example 2.

TABLE 3: The RMSE for Example 2.

	RMSE
e_{u1}^2	0.0288
e_{u2}^2	0.9423
e_y^2	0.0280
e_{x1}^2	9.3212×10^{-4}
e_{x2}^2	1.7209
e_{x3}^2	0.0766
e_{x4}^2	0.0523
e_{x5}^2	0.1324
e_{x6}^2	0.0162

the tower input is fed; consequently, current is not generated, the tower moves far of the maximum air intake, and the wind turbine blades do not have motion; (3) from 4 s to 6 s, the air is fed and the tower input is not fed; consequently, the tower does not move, the generator current is minimum, and the wind turbine blades almost do not move; (4) from 6 s to 8 s, the air is not fed and the tower input is fed with a negative voltage; consequently, current is not generated, the tower returns to the maximum air intake, and the wind turbine blades do not have motion. The dynamic model is a good approximation of the wind turbine behavior because

the signals of the first are near to the signals of the second. From Table 2, it is shown that the dynamic model is a good approximation of the real process because the RMSE is near to zero.

4. Conclusion

In this paper, a dynamic model of a wind turbine with a rotatory tower was introduced; the simulations using the parameters of a prototype showed that the proposed dynamic model has an acceptable approximation of the wind turbine behavior. The proposed dynamic model could be used on control, prediction, or classification. As a future research, the modeling will be improved using other interesting methods as the least squares in [16–26], neural networks in [14, 27–32], or fuzzy systems in [15, 33–35].

Conflict of Interests

The authors declare that there is no conflict of interests regarding the publication of this paper.

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